

# Regenerative Braking and Yaw Dynamics Optimal Control in Hybrid Vehicles



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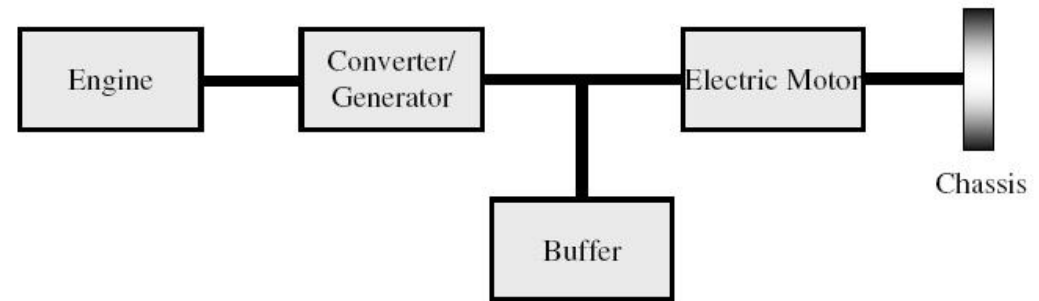
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*Active Safety Functions, Volvo Car Corporation,  
Göteborg, Sweden*

# Regenerative braking

## Hybrid vehicles

- Two possible energy flows in driving and braking



“Series Hybrid Vehicle”

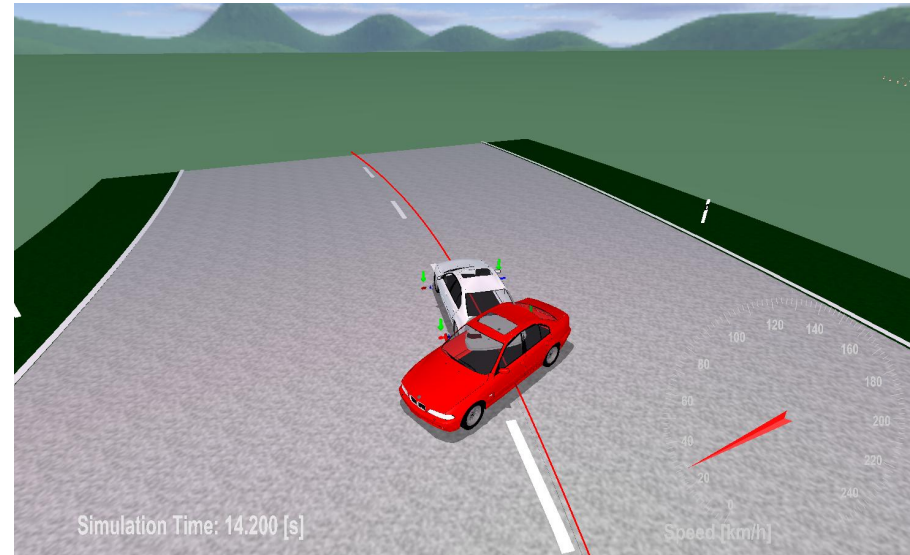
### *Regenerative braking*

Recovering energy by converting vehicle kinetic energy to electric energy



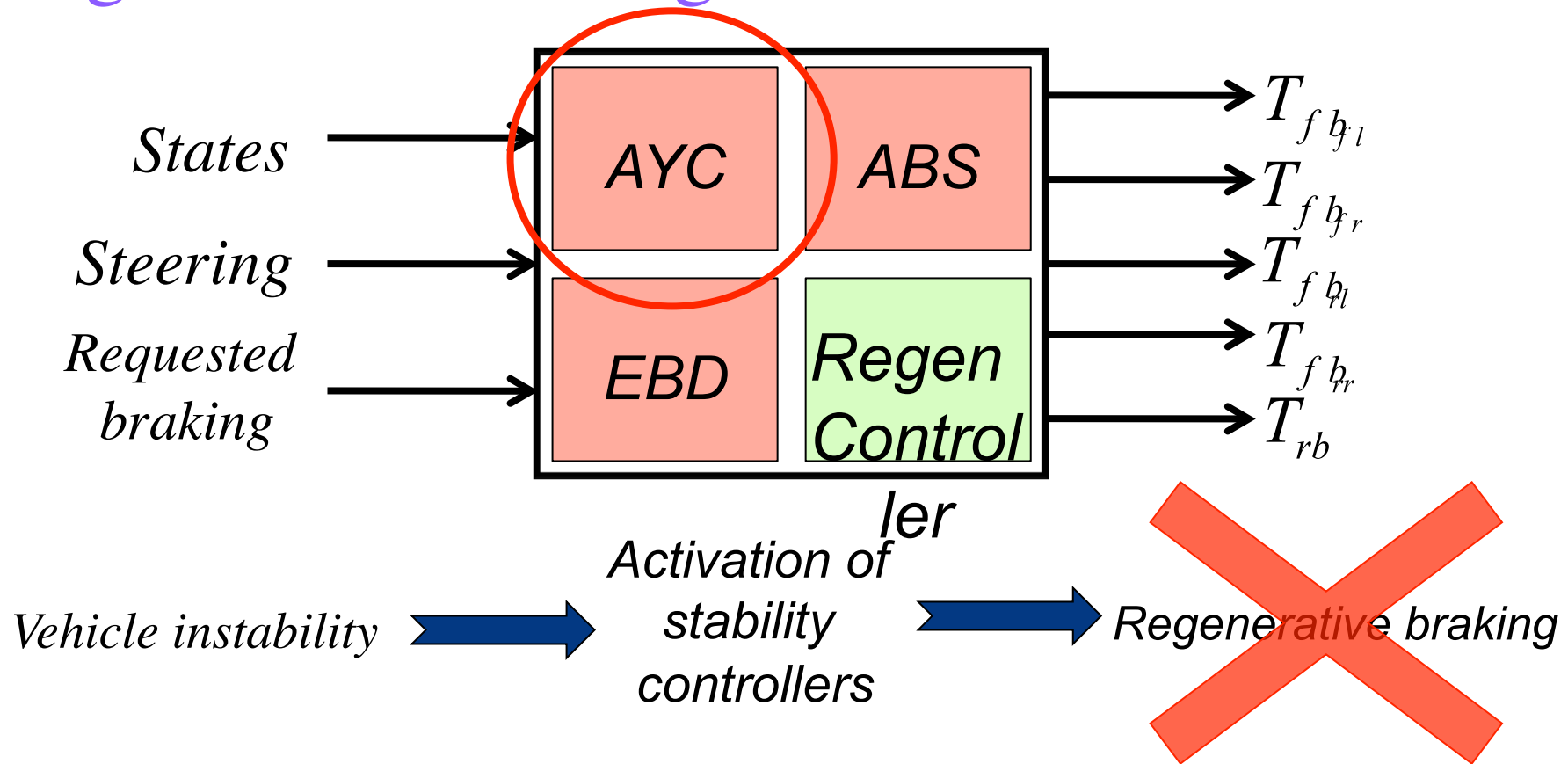
# Regenerative braking

Regenerative braking at the rear axle may induce instability on low- $\mu$  surfaces



Given a driver's braking request while *cornering on an icy surface*  
maximize the regenerative braking

# Regenerative braking



*Maximizing the amount of regenerative braking while preserving the vehicle stability and satisfying system constraints.*

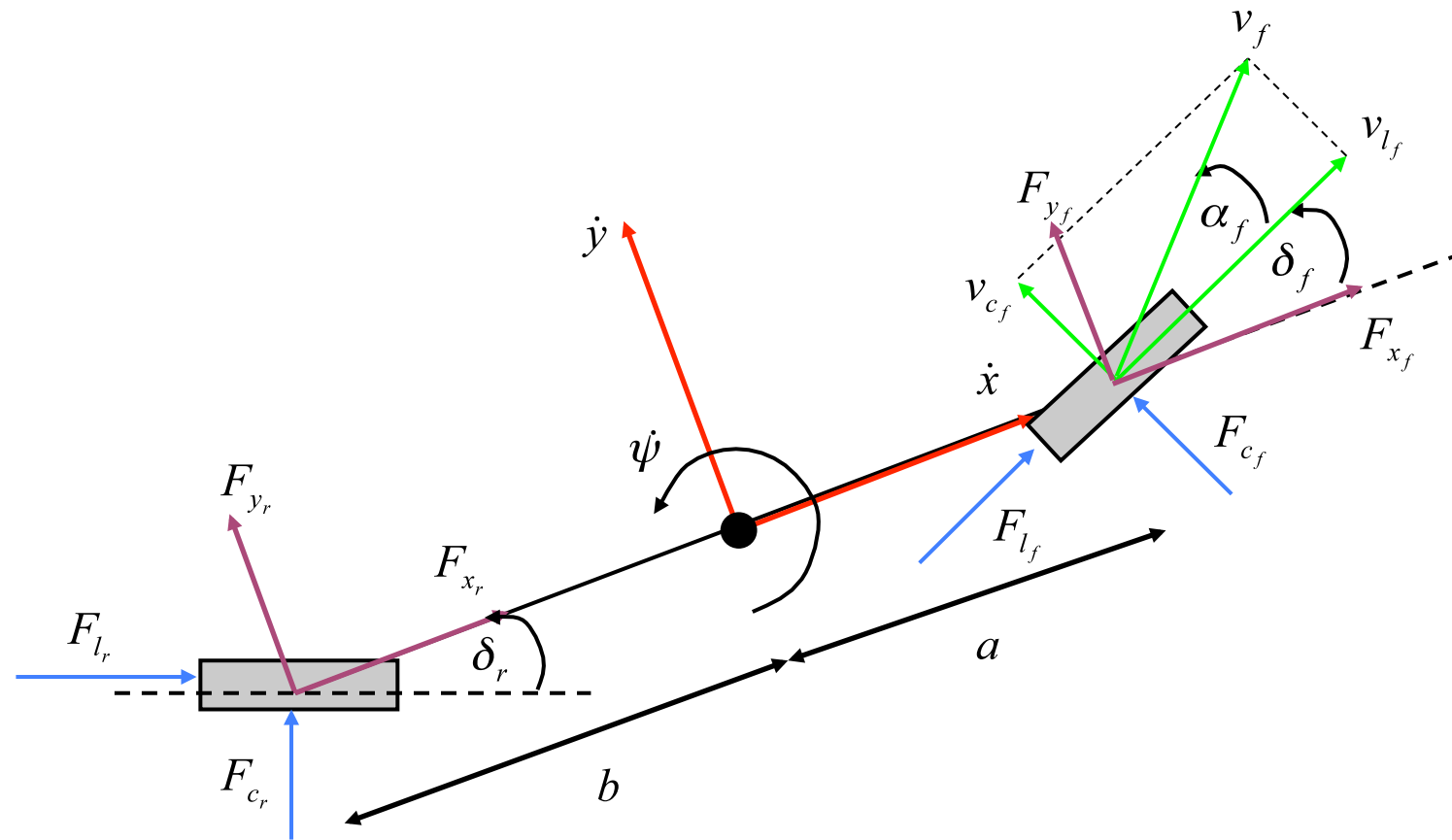
# Outline

- Vehicle modeling
  - Tire modeling
- Model predictive control design
  - Constraints and cost function definition
  - Problem formulation
- Simulation and experimental results
  - F/R braking shift in step  $\mu$
- Conclusions

# Outline

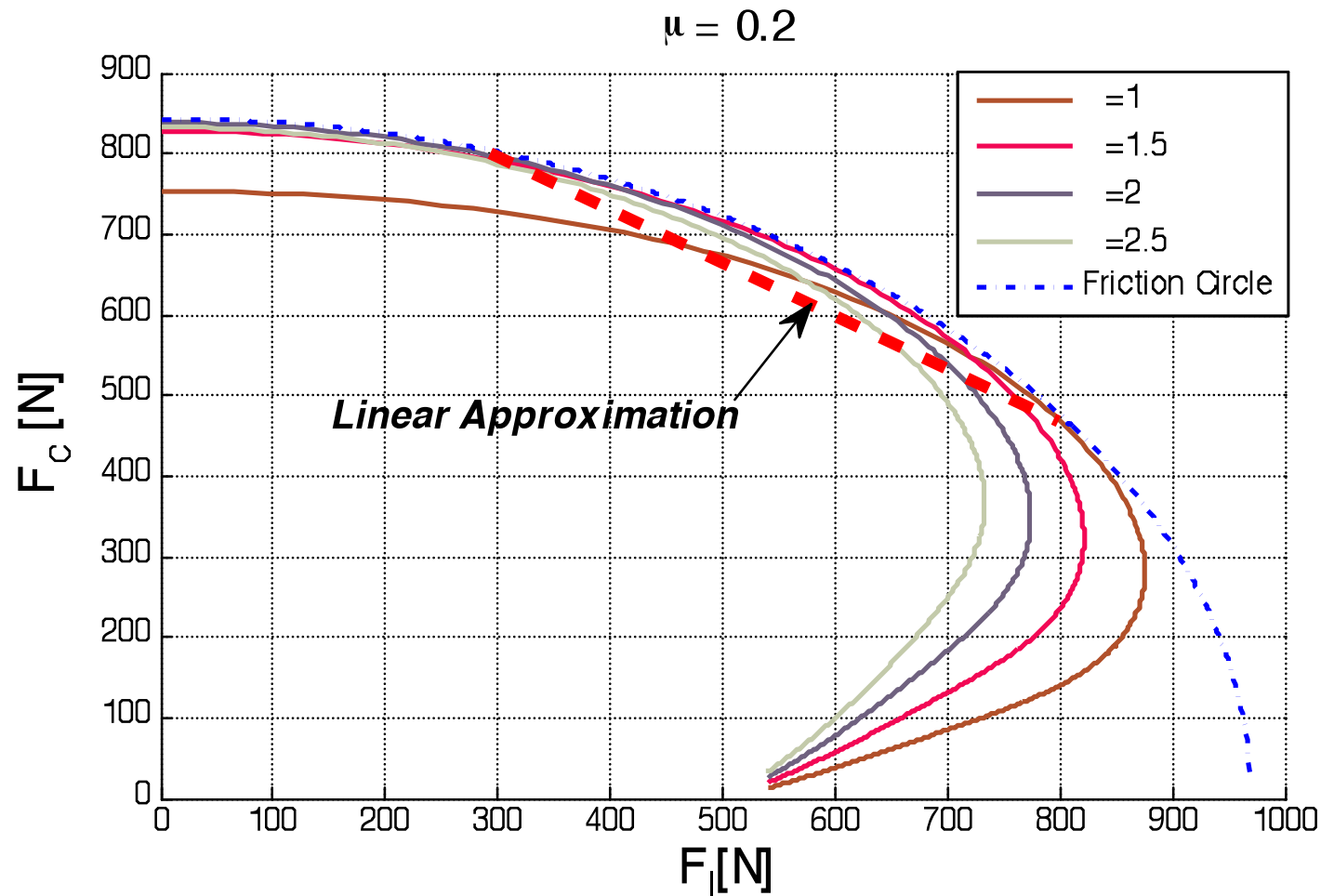
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# Vehicle modeling. Single track model



# Tire modeling

Piecewise Linear approximation :  $F_{c_{*},\bullet} = C_{c_{*},\bullet} \alpha_{*,\bullet} + D_{c_{*},\bullet} F_{l_{*},\bullet}$ ,  $D_{c_{*},\bullet} = \frac{\partial F_{c_{*},\bullet}}{\partial F_{l_{*},\bullet}}$ .





# Vehicle modeling

Simplified bicycle vehicle model

$$\dot{\xi}(t) = f(\xi(t), u(t), d(t)).$$

where

$$\xi = \begin{bmatrix} \dot{y} & \dot{x} & \dot{\psi} \end{bmatrix},$$

$$u = \begin{bmatrix} F_{fb_f} & F_{fb_r} & F_{rb} \end{bmatrix}, \quad F_{l_f, \cdot} = F_{fb_f, \cdot},$$

$$d = \delta_f. \quad F_{l_r, \cdot} = F_{fb_r, \cdot} + F_{rb}.$$

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# Model Predictive Control (MPC)

$$\min_U x(N)'Px(N) + \sum_{k=0}^{N-1} x(k)'Qx(k) + u(k)'Ru(k)$$

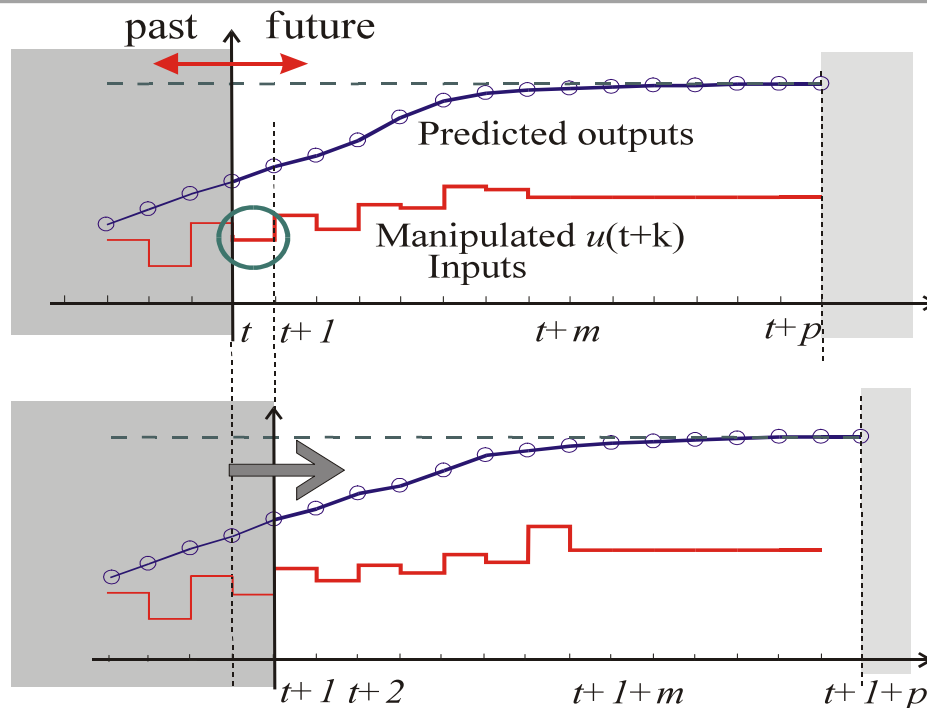
subject to  $x(k+1) = f(x(k), u(k))$

$$x(k) \in X, u(k) \in U$$

$$g(x(k), u(k)) = 0$$

**State and input constraints**

**Solved in Receding Horizon**



1. Optimize at time  $t$  (new measurements)
2. Only apply the first optimal move  $u(t)$
3. Repeat the whole optimization at time  $t+1$
4. Optimization using current measurements **Feedback**

# Control design

## Constraints:

- Vehicle Dynamics Constraints:

$$\xi_{k+1,t} = A_t \xi_{k,t} + B_t u_{k,t} + d_{k,t},$$

$$u_{k,t} = u_{k-1,t} + \Delta u_{k,t},$$

$$\eta_{k,t} = h(\xi(k)) = [0 \ 0 \ 1] \xi_{k,t}.$$

- Input constraints:

- Maximum regenerative braking.

- Total delivered force:  $F_D = F_{fb} + F_{rb}$

- Input rate.

- 70/30 constraint on friction braking:

$$F_{fb} = F_{fb_f} + F_{fb_r}, \text{ with } \begin{cases} F_{fb_f} = 0.7 F_{fb} \\ F_{fb_r} = 0.3 F_{fb} \end{cases}$$

# Control design

- Problem Formulation:

- Constraints:

- Constraint on the yaw rate tracking error:

$$E_{\min} - \varepsilon \leq \dot{\psi} - \dot{\psi}_{ref} \leq E_{\max} + \varepsilon$$

- Cost Function:

$$J(\xi_t, \Delta u_t, \varepsilon) = \sum_{i=1}^{H_p} \left\| \eta_{t+i,t} - \eta_{ref_{t+i,t}} \right\|_Q^2 + \sum_{i=0}^{H_u-1} \left\| \Delta u_{t+i,t} \right\|_R^2 \\ + \sum_{i=0}^{H_u-1} \left\| u_{t+i,t} \right\|_S^2 + \rho \varepsilon^2.$$

# Control design

- Problem Formulation:

$$\min_{\Delta u_t, \varepsilon} J(\xi_t, \Delta u_t, \varepsilon)$$

*subj.to*

Vehicle  
dynamics

$$\begin{aligned}\xi_{k+1,t} &= A_t \xi_{k,t} + B_t u_{k,t} + d_{k,t}, \\ \eta_{k,t} &= h(\xi(k)) = [0 \ 0 \ 1] \xi_{k,t}, \\ k &= t, \dots, t + H_p,\end{aligned}$$

Input  
constraints

$$\begin{aligned}u_{k,t} &= u_{k-1,t} + \Delta u_{k,t}, \\ u_{\min} &\leq u_{k,t} \leq u_{\max}, \\ F_{D_{k,t}} &= [1 \ 1 \ 1] u_{k,t}, \\ F_{f \ b} &= 2.3 F_{f \ b} \\ \Delta u_{\min} &\leq \Delta u_{k,t} \leq \Delta u_{\max},\end{aligned}$$

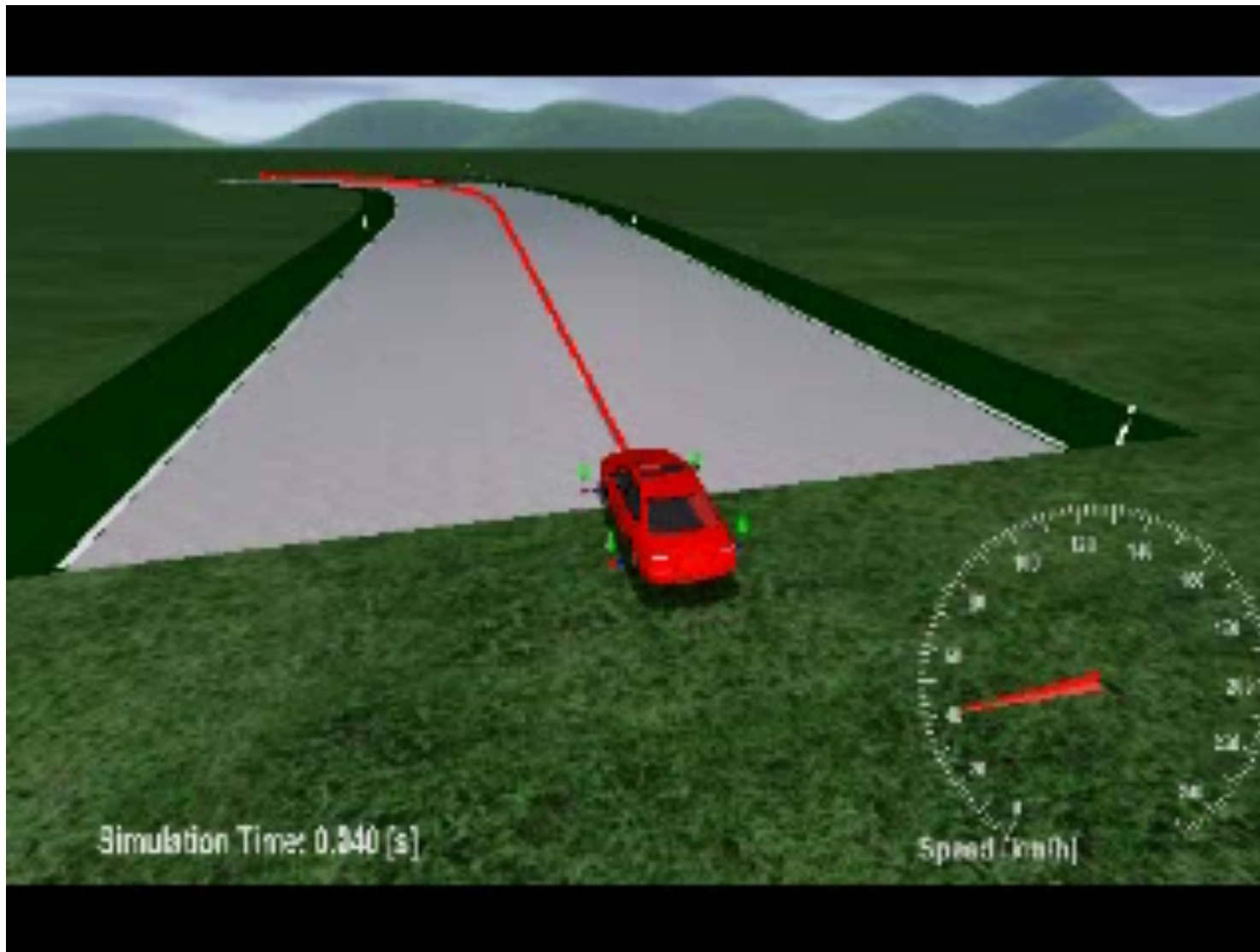
Additional  
constraints

$$\begin{aligned}A \Delta u_{k,t} &\leq b_1 + \varepsilon, \\ -A \Delta u_{k,t} &\leq -b_2 - \varepsilon, \\ k &= t, \dots, t + H_c - 1, \\ \xi_{t,t} &= \xi(t).\end{aligned}$$

# Outline

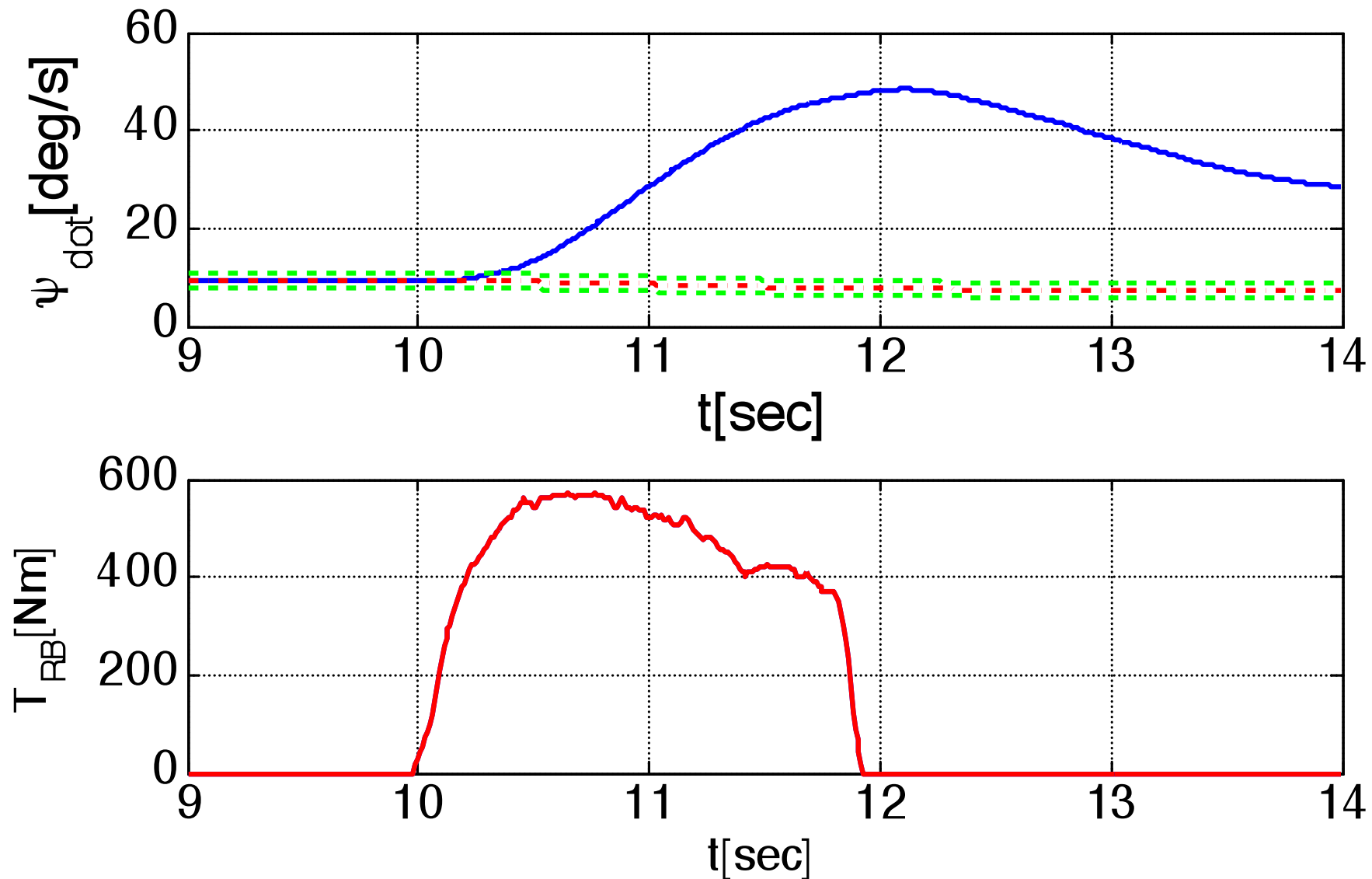
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# Simulation and Experimental Results:



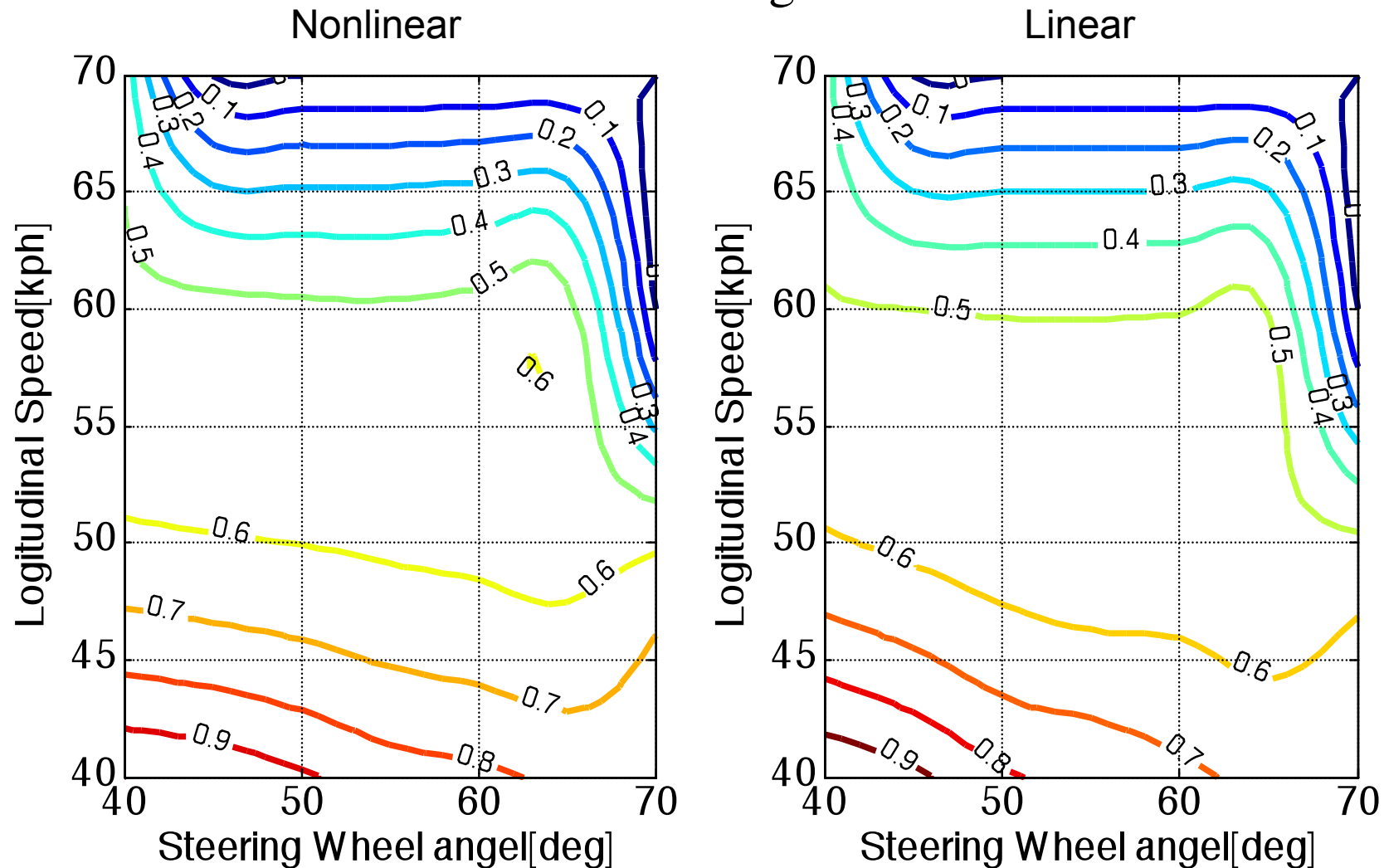


# Simulation and Experimental Results:



# Nonlinear vs linear MPC controllers

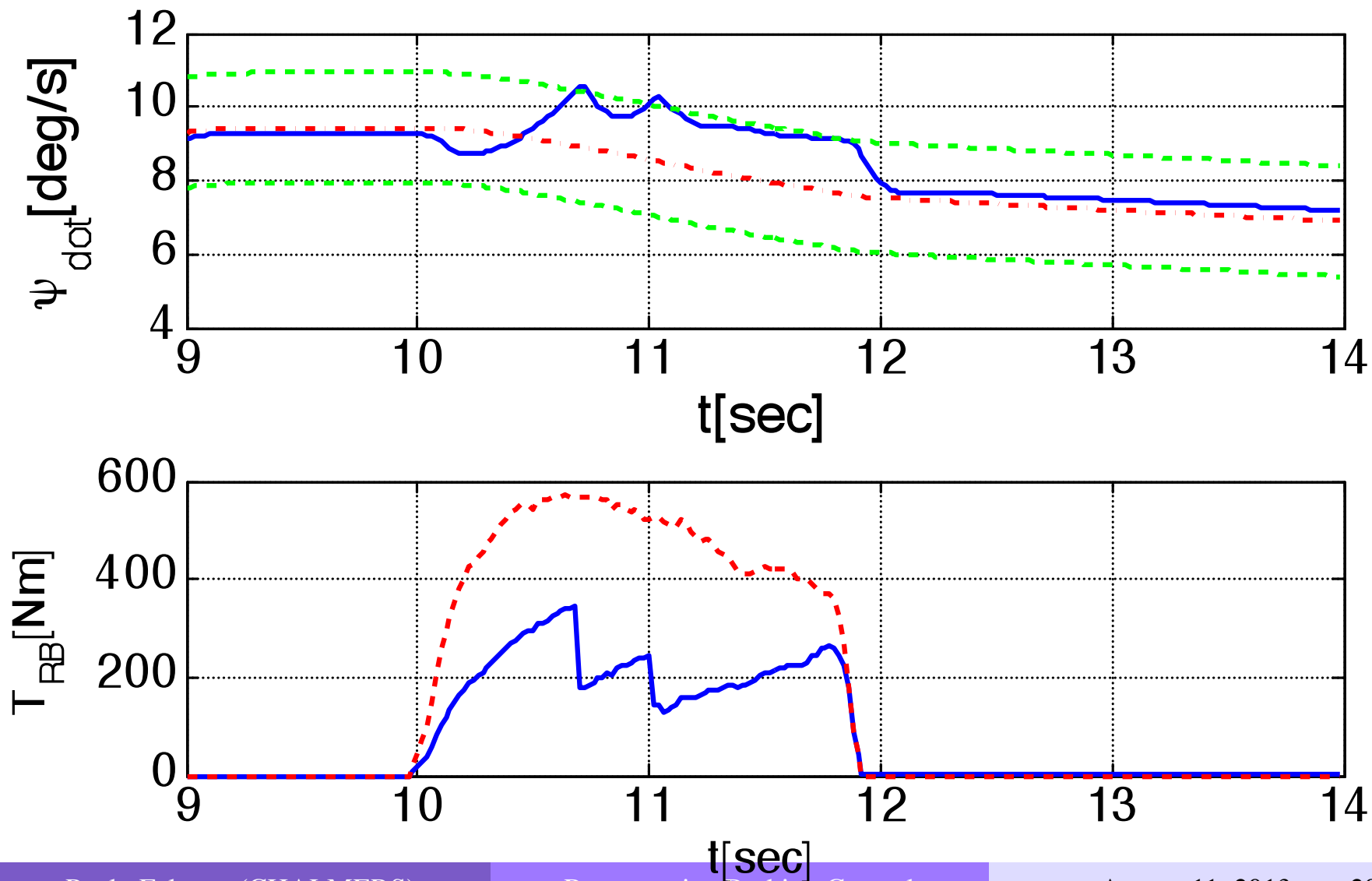
Percentage of requested braking delivered through regenerative braking



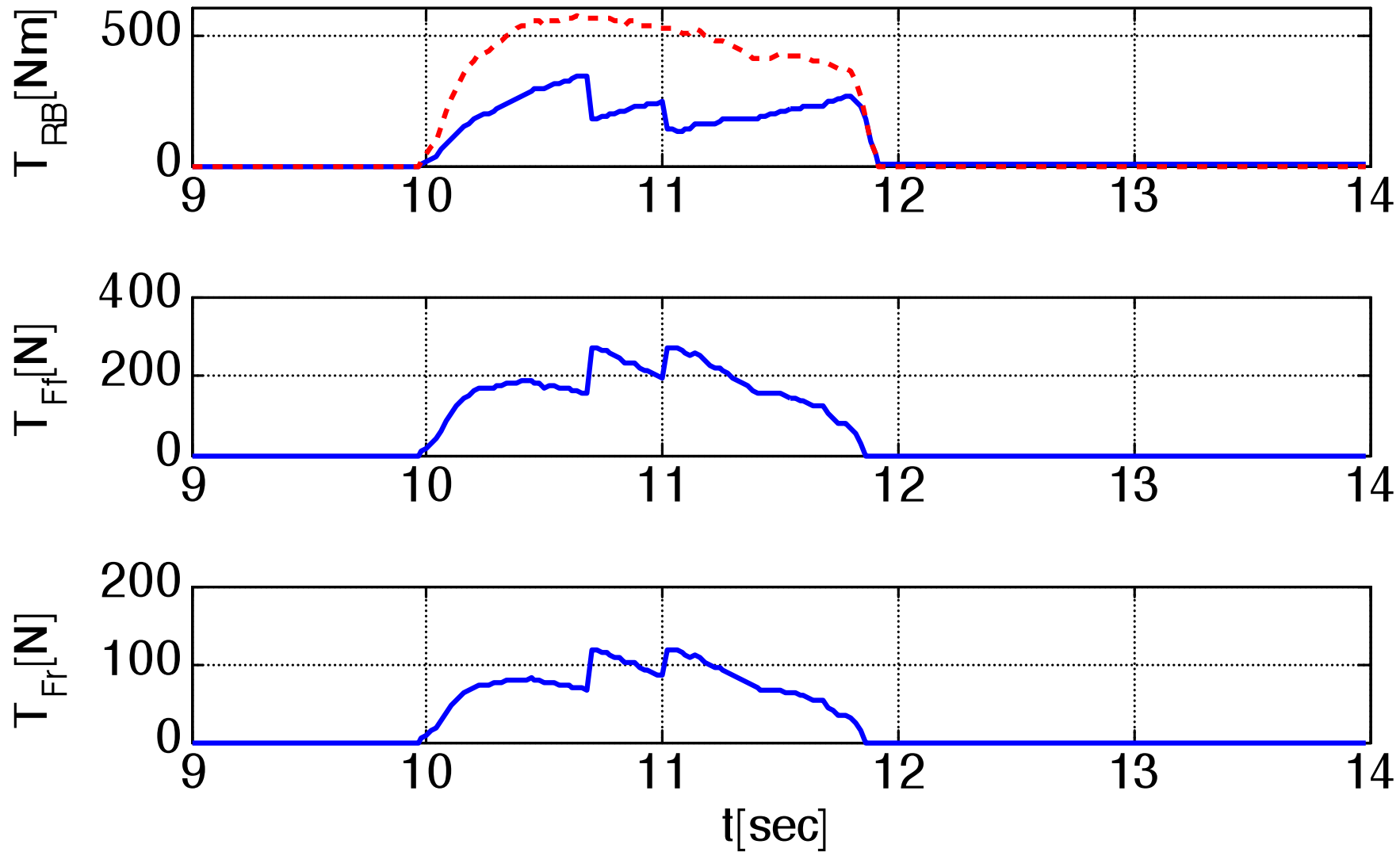
# Simulation results. Nonlinear bicycle model

Curve at 50 [Kph] with  
40 [deg] Steering wheel angle

## 50 [Kph]. Yaw rate and regen braking torque



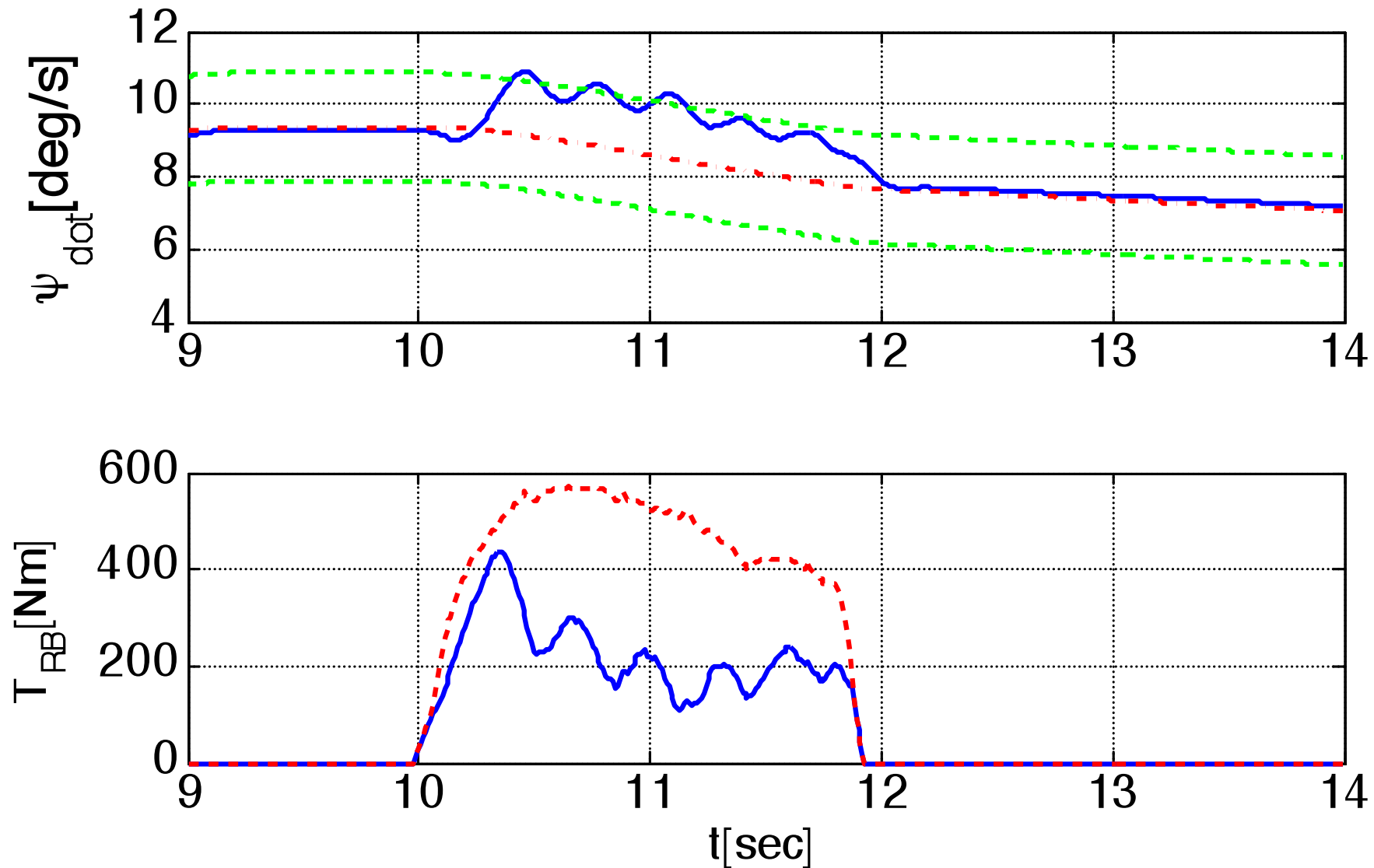
# 50 [Kph] Braking Torques



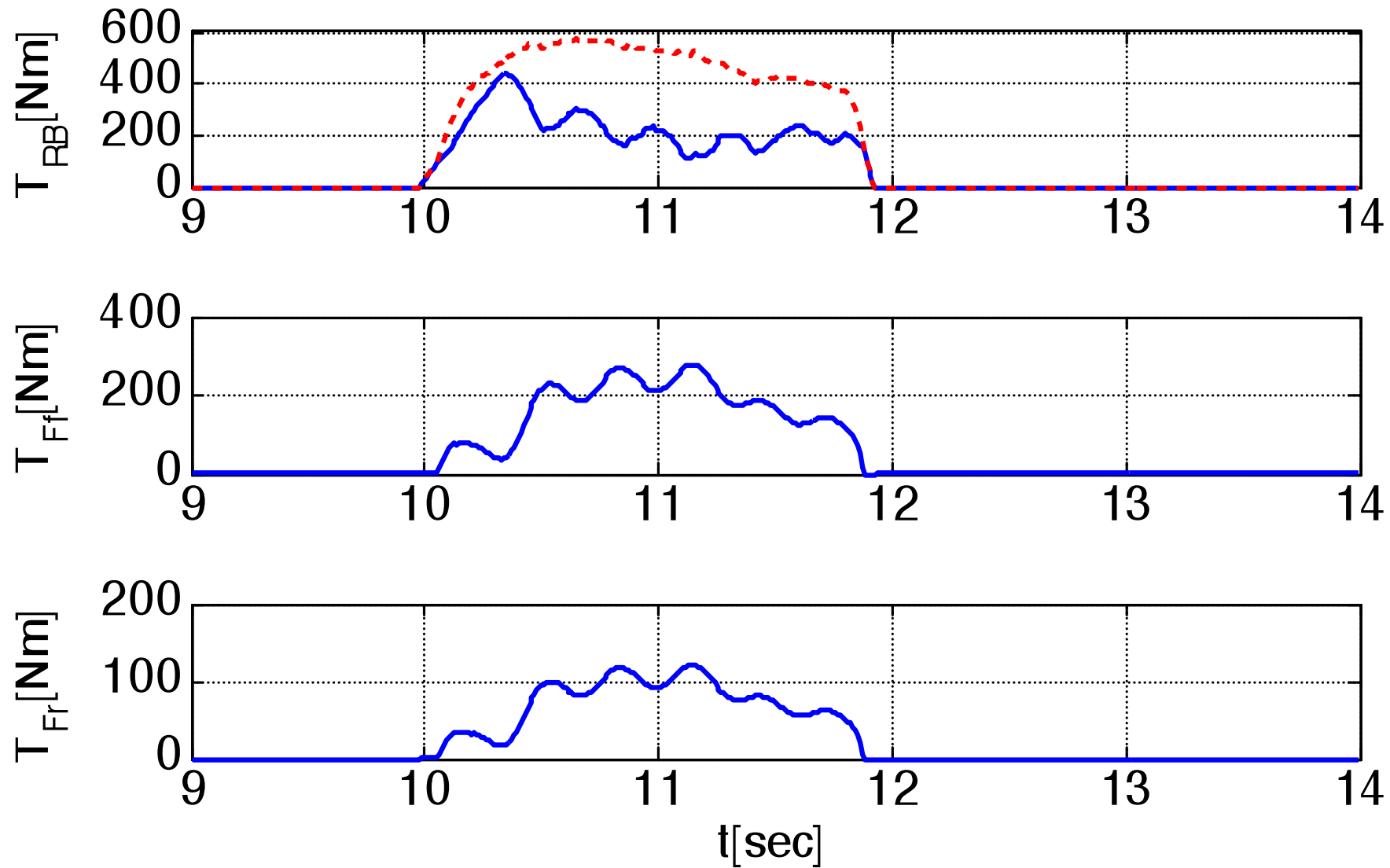
# Simulation results. Simplified bicycle model

Curve at 50 [Kph] with  
40 [deg] Steering wheel angle

## 50 [Kph]. Yaw rate and regen braking torque



## 50 [Kph]. Braking torques

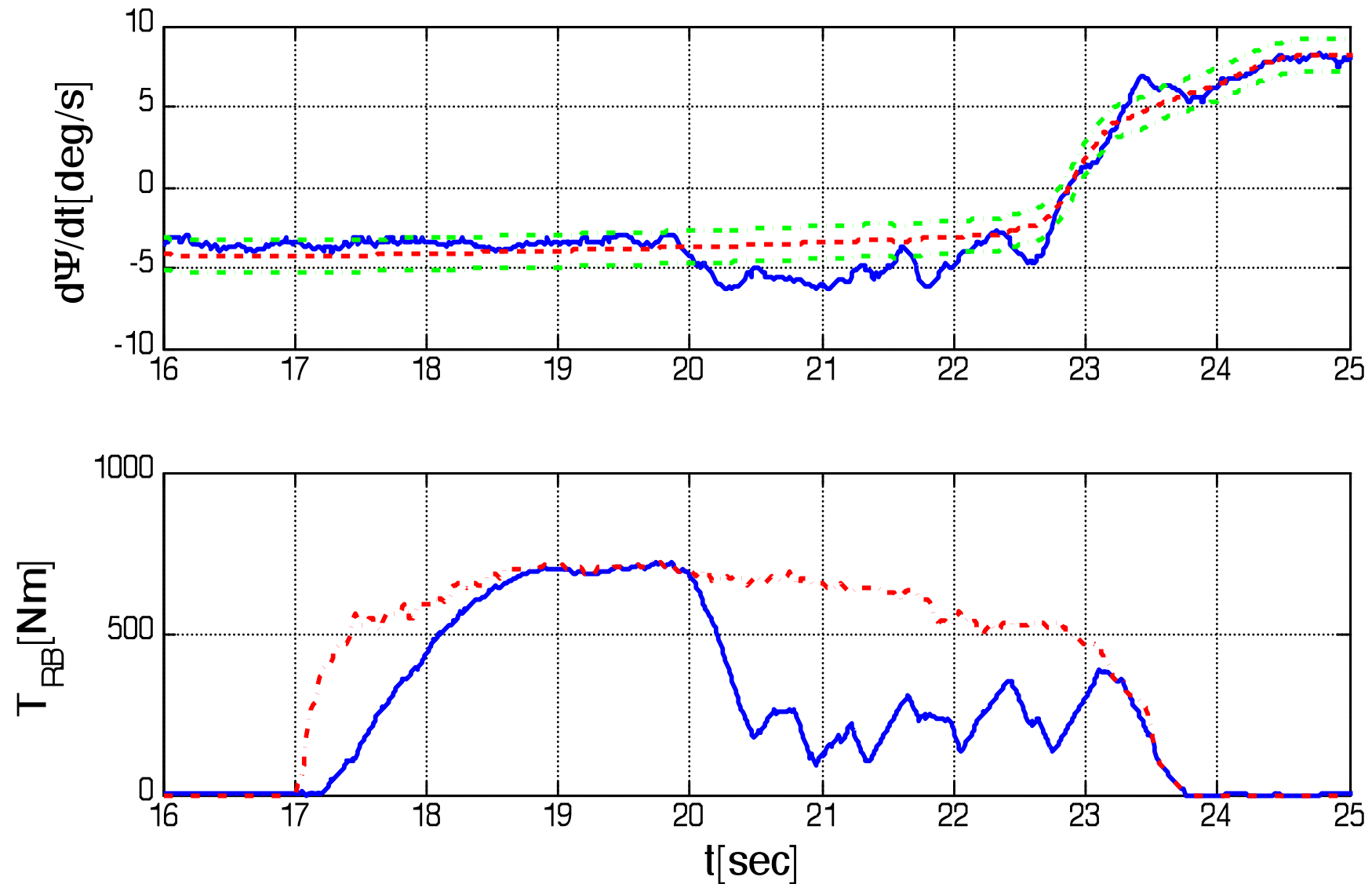




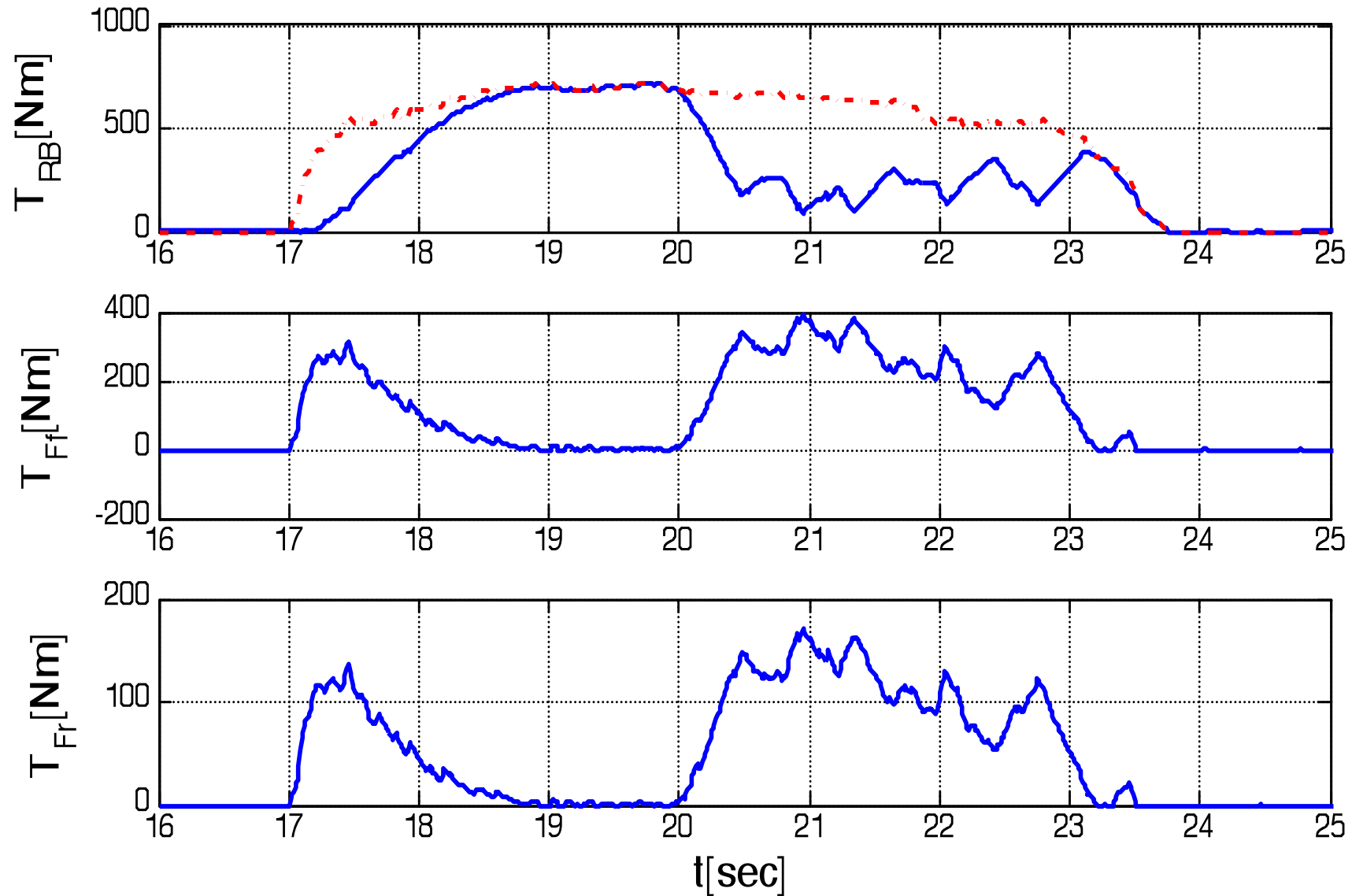
# Experimental results

Curve at 60 [Kph] with  
50 [deg] Steering wheel angle

## 60 [Kph]. Yaw rate and regen braking torque



# 60 [Kph] Braking Torques



# Conclusions

- Model based approach to regenerative braking control
- Regenerative braking optimized while preserving vehicle stability
- Higher priority active safety systems activation minimized